

DISCONNECTABLE HEAT EXCHANGER

Related Art

The invention is based on a heat exchanger according to the preamble of Claim 1.

The specific contamination of the environment resulting from the emission of carbon dioxide by internal combustion engines is primarily a factor of their efficiency. This is not satisfactory, among other things, when the internal combustion engine is operated below its optimal operating temperature. In a few operating conditions, such as during cold starting or long downhill driving of motor vehicles, the optimal temperature of the internal combustion engine is not reached, which leads to increased fuel consumption and increased exhaust-gas emissions. During cold starting at low outside temperatures, heat is also needed to deice the windows or to warm the passenger compartment of the vehicle, in order to improve driving safety and comfort. Currently, chemical or electrical supplementary heating systems are used in particular to solve this problem. Their use also results in increased fuel consumption, however.

A heat exchanger was made known in a supplement entitled "System Partners 98", page 4 of the MTZ (Motor Technische Zeitschrift) 7/2 98 that is installed in an exhaust-gas return line in order to cool the exhaust gas to be returned to a combustion chamber of the internal combustion engine. Cooling the exhaust gas improves the aspiration. Since the heat exchanger is used constantly during the entire operation of the internal combustion engine, heat is continuously absorbed by the coolant of the internal combustion engine, even during full-load operation. In order to prevent overheating of the coolant and the internal combustion engine in this driving state, however, heat must also be dissipated by way of the main

09/913562 101301

1 radiator of the internal combustion engine. The main radiator and the cooling
2 fans assigned to it must be designed accordingly to be larger in size.

4 Advantages of the Invention

6 According to the invention, the heat exchanger is arranged in a main exhaust-gas
7 flow, and a shutoff device is provided in the coolant inflow. As a result, the usable
8 heat of the exhaust air can be used optimally during cold starting, in order to
9 reach the operating temperature of the internal combustion engine as rapidly as
10 possible and to use a sufficient quantity of heat for deicing the vehicle windows,
11 and to warm up the passenger compartment. In his dissertation entitled
12 "Optimization of Fuel Consumption, Emissions, and Heating Comfort in Diesel
13 Vehicles Using Energy Flow Management", E.D. Pott states that the usable heat
14 of the exhaust gas amounts to approximately 1.4 KW in just one driving cycle
15 even in a smaller Diesel internal combustion engine for a passenger car. In a
16 passenger car with an internal combustion engine that functions according to the
17 Otto principle, the usable heat is markedly greater due to higher exhaust-gas
18 temperatures.

20 When the internal combustion engine has reached its optimal operating
21 temperature, a shutoff valve closes the coolant inflow, thereby interrupting the
22 passage of coolant through the heat exchanger, so that the main radiator of the
23 internal combustion engine and the blower assigned to it do not need to be
24 designed to be larger in size.

26 So that the coolant remaining in the heat exchanger when the coolant inflow is
27 shut off does not overheat and thereby decompose and cause deposits in the
28 coolant ducts of the heat exchanger, it is appropriate to displace the coolant out
29 of the coolant ducts as soon as the shutoff device in the coolant inflow is closed.
30 The coolant is returned to the coolant ducts shortly before the shutoff device is
31 reopened. To this end, a gas reservoir is connected at a high point of the coolant

1 ducts, from which gas, usually air, is fed into the coolant ducts and later
2 removed.

3

4 The gas reservoir is designed in simple fashion as a bellows, on one face of
5 which a connecting line leads to the coolant ducts and on the opposite face of
6 which an actuator acts. This shortens the bellows and thereby presses a
7 corresponding volume of gas through the connecting line into the coolant ducts.
8 The actuator can be operated electrically, hydraulically, and/or pneumatically.
9 When the actuator is reset, the bellows expands again and draws the air out of
10 the coolant ducts.

11

12 As an alternative to this, a bypass line is provided between the exhaust-gas inlet
13 and the exhaust-gas outlet, on the branch of which a shutoff device is arranged,
14 in order to control the exhaust-gas inlet and the exhaust-gas outlet in
15 complementary fashion. When the shutoff device closes the coolant inflow, the
16 shutoff device at the branch of the bypass line simultaneously shuts off the
17 exhaust-gas inlet and opens the bypass line. Since exhaust gas no longer
18 passes through the heat exchanger now, overheating of the coolant is reliably
19 avoided.

20

21 Basically, the heat exchanger can be arranged in any suitable location in a main
22 exhaust-gas stream of a vehicle. Appropriately, however, it is arranged behind a
23 catalytic exhaust-gas converter in order to prevent the catalytic exhaust-gas
24 converter from being delayed in reaching its operating temperature.

25

26

Drawing

27

28 Further advantages arise out of the following drawing description. Embodiments
29 of the invention are shown in the drawing. The drawing, the description, and the
30 claims contain numerous features in combination. It is appropriate for the expert

0913562-101801

1 to also examine the features individually and combine them into additional logical
2 combinations.

3

4 The sole figure shows a schematic representation of a disconnectable heat
5 exchanger.

6

7

Description of the Design Examples

8

9 The heat exchanger 10 shown functions according to the countercurrent principle
10 and is arranged in a main exhaust-gas flow 34. It has an exhaust-gas inlet 30
11 and an exhaust-gas outlet 32 that are connected with each other by way of
12 exhaust-gas ducts 36. Coolant ducts 14, which are connected with a coolant
13 inflow 26 and a coolant return 28, are adjacent to the exhaust-gas ducts 36. The
14 coolant flow is indicated by arrows 38 and 40.

15

16 A shutoff device 20 is provided in the coolant inflow 26, which restricts or closes
17 the coolant inflow 26 more or less as a function of the operating and ambient
18 parameters. The shutoff device 20 is controlled by an electronic control device 12
19 by way of a signal line 42. This can be an integrated component of engine
20 electronics.

21

22 The coolant ducts 14 are connected with a gas reservoir 16 by way of a
23 connecting line 18 at a high point 24, which gas reservoir 16 is designed as a
24 bellows and can be changed in its length between lines 52 and 54 by way of an
25 actuator. While the connecting line 18 is provided at the one face 48 of the gas
26 reservoir 16, the actuator acts on the opposite face 50. The actuator 22 is also
27 controlled by the electronic control unit 12 by way of a signal line 44.

28

29 When the shutoff device 20 is closed, the actuator 22 begins to function and
30 shortens the bellows 16. As a result, the interior space 46 of the bellows 16
31 becomes smaller, so that the gas, usually air, is fed into the coolant ducts 14 by

0933552-104004

1 way of the connecting line 18 and displaces the coolant there. The coolant can
2 therefore not be heated any further by the exhaust gasses.

3
4 Shortly before the shutoff device 20 reopens, the actuator 22 returns to its
5 starting position, whereby it draws the gas from the coolant ducts 14 back into
6 the expanding interior space 46 of the bellows 16. When the passage through the
7 heat exchanger 10 is restored, the coolant can therefore absorb heat from the
8 exhaust air.

9
10 An alternative to this is illustrated using dotted lines. In this alternative, the
11 exhaust-gas inlet 30 is connected with the exhaust-gas outlet 32 by way of a
12 bypass line 56. A further shutoff device 58 is provided in the area of the branch of
13 the bypass line 56, which is connected with the control unit 12 by way of a signal
14 line 60 and controls the exhaust-gas inlet 30 and the bypass line 56 in
15 complementary fashion, i.e., the exhaust-gas inlet 30 is restricted or closed that
16 much more, the more the bypass line 56 is opened.

17
18 If the shutoff device 20 at the coolant inflow 26 is now closed, the shutoff device
19 58 closes the exhaust-gas inlet 30 and opens the bypass line 56 almost
20 simultaneously. The exhaust gas is thereby directed past the heat exchanger 10,
21 so that the coolant in the coolant ducts 14 cannot overheat. When the shutoff
22 device 20 opens, the shutoff device 58 also opens the exhaust-gas inlet 30 and
23 closes the bypass line 56.

24
25 In order to not disrupt the operating behavior of a catalytic exhaust-gas
26 converter, which is not shown in greater detail, it is appropriate to arrange the
27 heat exchanger 10 downstream from the catalytic exhaust-gas converter.
28

0913552-101801

Reference Symbols

1

2

10	Heat exchanger	54	Line
12	Control unit	56	Bypass line
14	Coolant duct	58	Shutoff device
16	Gas reservoir	60	Signal line
18	Connecting line		
20	Shutoff device		
22	Actuator		
24	High point		
26	Coolant inflow		
28	Coolant return		
30	Exhaust-gas inlet		
32	Exhaust-gas outlet		
34	Main exhaust-gas flow		
36	Exhaust-gas duct		
38	Arrow		
40	Arrow		
42	Signal line		
44	Signal line		
46	Interior space		
48	Face		
50	Face		
52	Line		

3

00013562 101801
 108101 2955E660